



Beyond Symbology: Mathematical Models for GD&T

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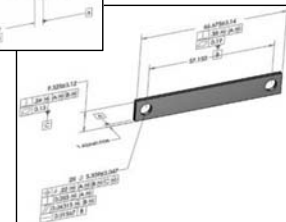
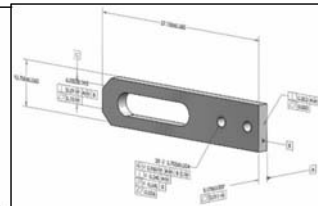
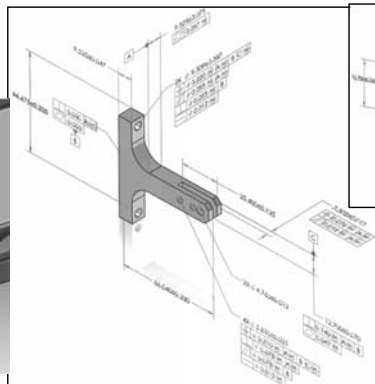
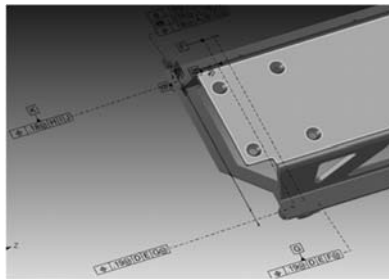
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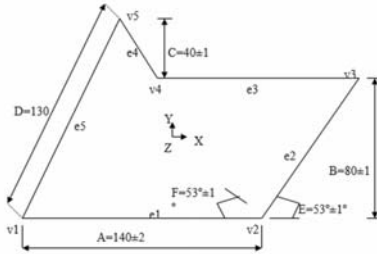
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Geometric Variability in Manufacturing



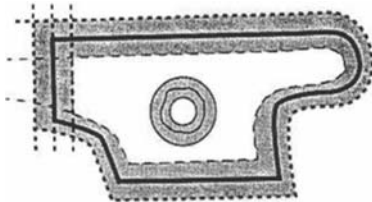
- Geometric dimensions & tolerances are of concern in all aspects of product development.
- Miscommunication and misinterpretation can result in low acceptance rates or expensive rework.
- Math models can not only prevent such problems but also provide a better medium for PMI information transfer between modern digital tools: CAD, CAM, CAE, CMM...



PARAMETRIC MODELS

Closely related to parametric CAD: uses the same set of parameters and constraints as those used in geometry construction; Tolerances are +/- variations applied to dimensions; the math model is the constraint set
 Cannot support form tolerances, datum ref frames, directional relations,...

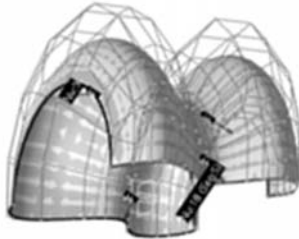
OFFSET ZONE MODELS



Tolerance zone created by Boolean subtraction of volumes obtained by offsetting a part's boundaries by equal amounts on either side

Cannot distinguish between variation types (size, form, orientation, position)

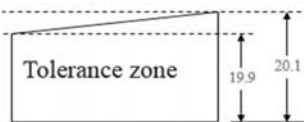
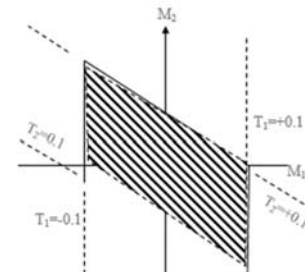
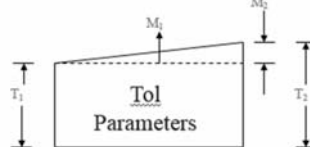
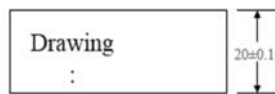
No DRFs, material or other modifiers



VARIATIONAL SURFACES

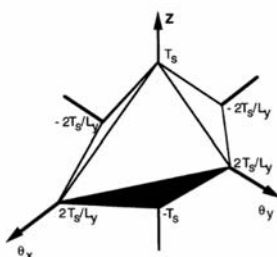
NURBS/B-spline Control Points are assigned tolerance values
 Non-intuitive- no explicit relation between CPs and GD&T
 Cannot differentiate between different tolerance classes
 No DRFs, modifiers, zones

The basic idea is to map geometric variation parameters to a hypothetical n-dimensional Euclidean space



Turner's 2D vector model of size:

The seed of this idea can be found in Josh Turner' work from the 90's; but it was not pursued further



Whitney's 3D vectorial tolerance model for size
 Max boundary in a "kinematic parameter space"



Topological Model: DoF Algebra



- DRFs and TRFs are clusters of points, lines and planes with different geometric relations to each other (coincident, //, \perp , ...)
- DoF Algebra includes symbolic ops to determine free and invariant DoFs of entity clusters.
- This algebra was validated by applying it to all cases in the Y14.5.1.

	Point	Line	Plane
X_{fdof}	$(T_x T_y T_z, \dots) \rightarrow (111,000)$	$(T_x T_y -, R_x R_y -) \rightarrow (110,110)$	$(- - T_z R_x R_y -) \rightarrow (001,110)$
X_{inv}	$(\dots, R_x R_y R_z) \rightarrow (000,111)$	$(- - T_z \dots R_z) \rightarrow (001,001)$	$(T_x T_y -, - R_z) \rightarrow (110,001)$

Coordinate transformations

We define the CS Transform operator $OP_{i>j}$ as swapping of the i and j element values for both TDof and RDof.
Thus, $OP_{z>y} \{ [110,110] \} = [101,101]$.
This is valid for any entity or cluster and for active or invariant DoF vector.



Algebraic Operators



Union Operation \cup			Intersection Op \cap		
a_i	b_i	$a_i \cup b_i$	a_i	b_i	$a_i \cap b_i$
1	1	1	1	1	1
1	0	1	1	0	0
0	1	1	0	1	0
0	0	0	0	0	0

Combining DoFs for clusters

$$[X_{fdof}] = [A_{fdof}] \cup [B_{fdof}];$$

$$[X_{inv}] = [A_{inv}] \cap [B_{inv}]$$

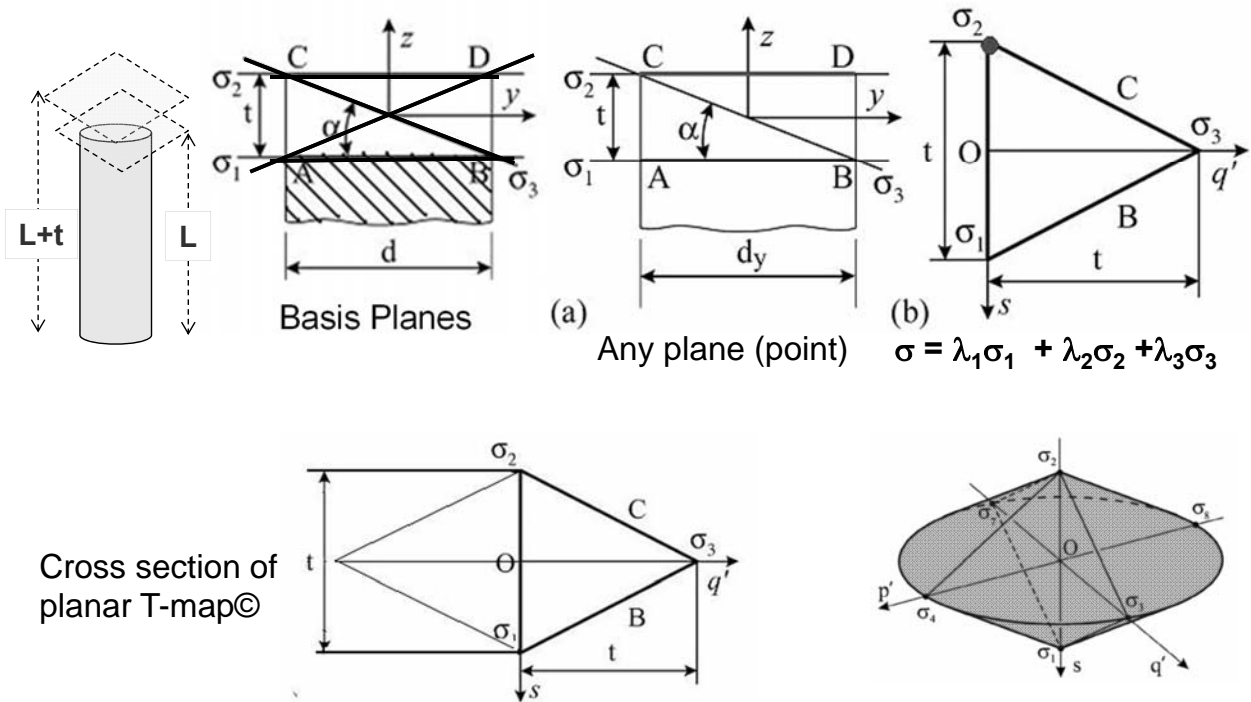
Algebraic Relations

- $[A] \cup [B] = [B] \cup [A]$ Commutative relation
- $[A_{fdof}] \cap [A_{inv}] = [\emptyset] = [000,000]$ Null set
- $[A_{fdof}] \cup [A_{inv}] = [I] = [111,111]$ “Identity” vector
- $[A_{inv}] = RCP \{ [A_{fdof}] \}$ Reciprocal relation (or \bar{A})
- +Standard Associative, Distributive and Idempotence relations

- DoF algebra models datum flow chains, DRF combinations and tolerance classes
- The controlled DOFs are the intersection of the DOFs of three tolerance elements.
- No matter what the target cluster is, the DOF vector of target entity is one of six combinations.

No.	Target	DRFs	Tol. Class	Constrained DOFs
1	(111,000)	(111,000)	(111,111)	(111,000)
		(111,110)		
		(111,111)		
		(110,110)		
6	(111,111)	(111,111)	(000,111)	(000,110)
		(110,110)		
		(001,110)		
		(111,111)		

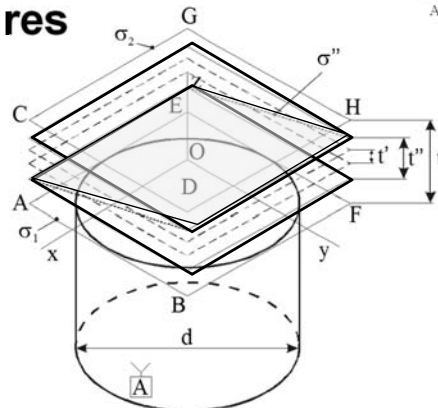
Cylindrical bar cross-sections



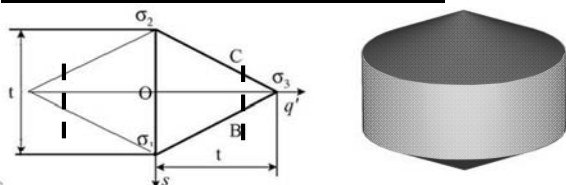
FLOATING ZONES

ORIENTATION zone (t'') translates $\uparrow\downarrow$
can rotate about x - or y -axes

FORM zone (t') translates $\uparrow\downarrow$
and rotates about x - or y -axes

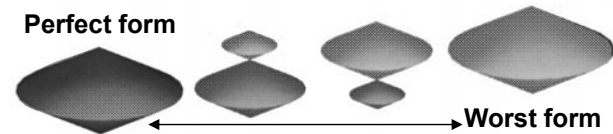


SIZE + ORIENTATION T-map



Addition of orientation tol t'' to size reduces the allowable tilt
Orientation T-map can be obtained from size by truncating the σ_3 axis

SIZE + FORM T-map

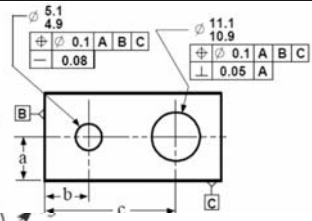


As per Y14.5 Rule#1

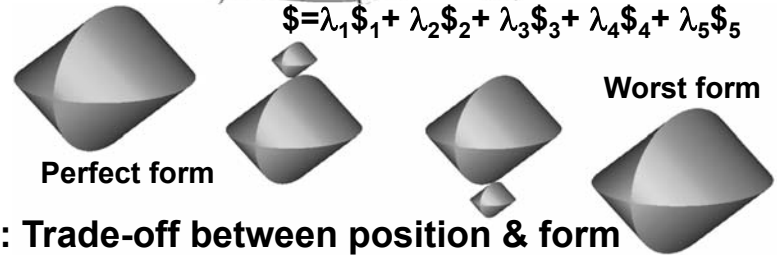
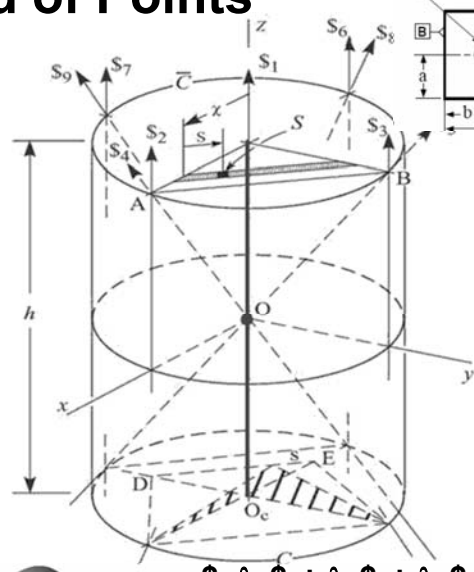
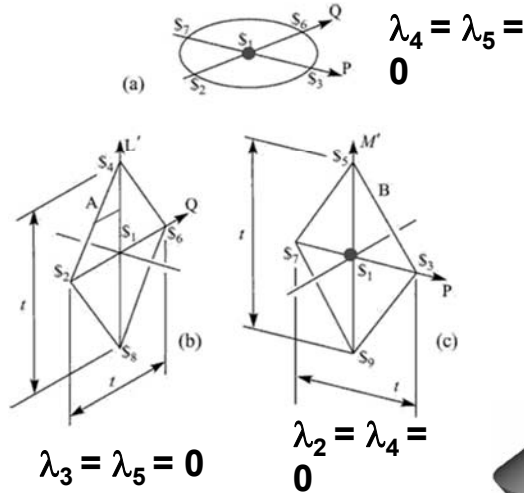
- Worst form occupies the entire zone
 - Perfect form occupies none
- Therefore, size + form is modeled by splitting into two planar T-maps that together must conform to size map



Tolerance Maps For Lines: 4-D Solid of Points



2D cross-sections
of the T-Map



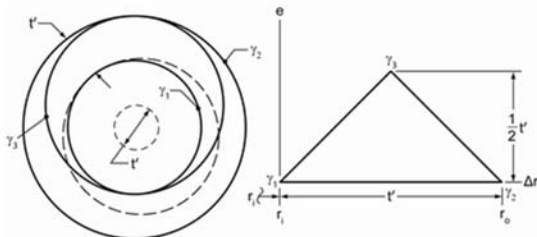
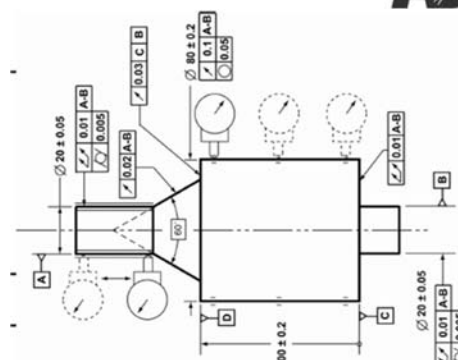
3D cross-sections: Trade-off between position & form



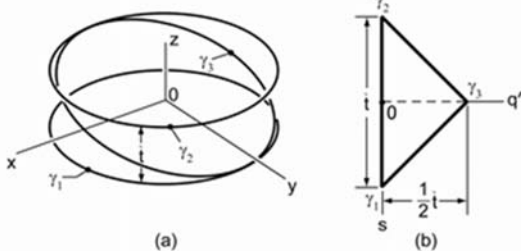
Circular Runout Model



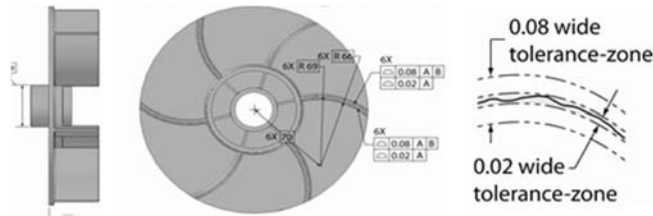
- Circular runout is a composite tolerance that controls both circularity and concentricity (position), independent of size
- Applied to any axisymmetric X-sec



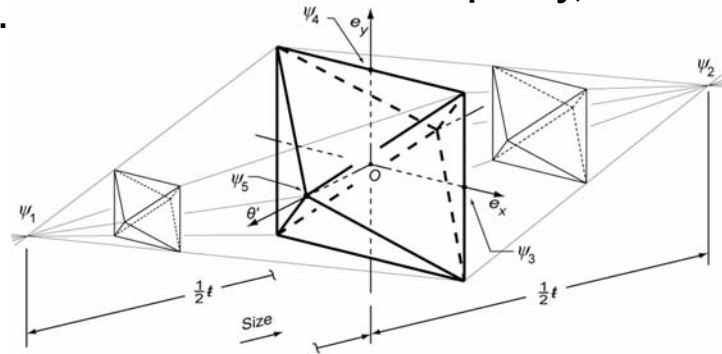
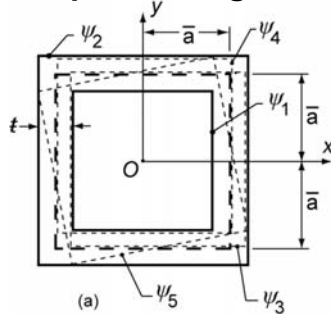
- circular X-sec, involves two variables: circularity (annular zone) + eccentricity
- (a) An annular tolerance-zone of amount t' which lies between the inner and outer boundaries γ_1 and γ_2 of radii r_i and r_o , respectively. (b) Its 2D T-Map



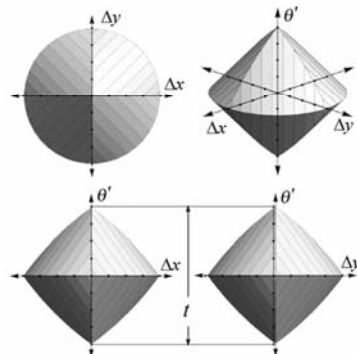
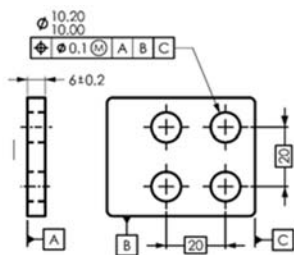
- planar (end) involves two variables: linear offset + angle
- (a) A cylindrical tolerance-zone of height t which lies between the upper and lower boundaries of γ_1 and γ_2 . (b) Its 2D T-Map.



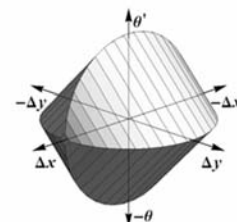
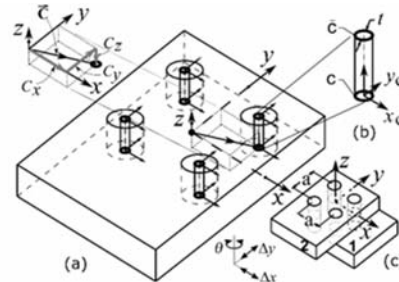
- Profile tolerances control the shape, size, and position of complex features, e.g. turbine blades and pump vanes.
- For line profiles, four variables are required to identify a variation of the theoretical shape within its tolerance-zone.
- **Example: A square line-profile**
- For line profiles, each point in the T-Map represents one square with a given size and x-, y-, and θ -position in the tolerance-zone. Consequently, the T-Map is a 4-D geometric shape.



- For each part, the T-Map that models limits to relative displacements between hole patterns formed by transforming the T-Map for each individual feature in its local frame of reference to a global frame that is at the center of the pattern.



Part-level T-Map for a circular (square) pattern of four holes



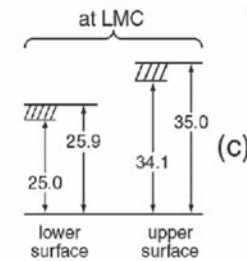
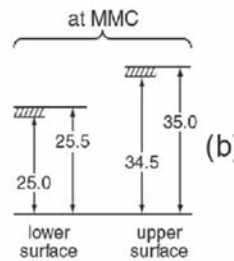
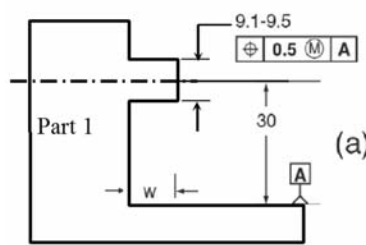
Part-level T-Map for opposite holes, represented in frame xyz.

More than 50 T-map models have been developed so far based on combinations of target feature, tolerance type and datum type

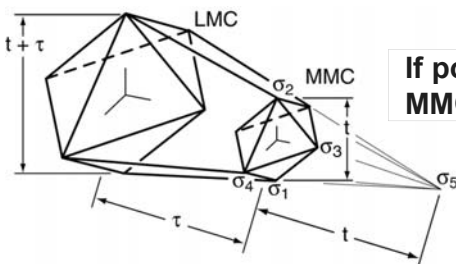
T-map	Geometry, tolerance, datum	T-map	Geometry, tolerance, datum
	Geom: Rect bar; plane Tol class: size Datum: none		Geom: Rect bar; plane Tol class: size + orient Datum: planar face
	Geom: Round bar; plane Tol class: size Datum: none		Geom: Round bar; plane Tol class: size + orient Datum: offset axis
	Geom: Round bar; plane Tol class: size + orient Datum: planar face		Geom: Planar circular face Tol class: circular runout Datum: axis
	Geom: traing bar; plane Tol class: size Datum: none		Geom: Rect bar; plane Tol class: size + orient Datum: two datums

Material Modifiers in T-map models

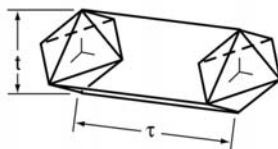
t = pos. tol
τ = size tol.



4-D T-Maps: size is the 4th dimension
The dipyramid now is the T-map for position of the medial plane.



If pos tol uses MMC modifier



If pos tol uses RFS modifier

Hyper-Volume computation
Hyperpyramid of dimension n

$${}_n V = \frac{1}{n} ({}_{n-1} C) h$$

$${}_4 V_f = \frac{1}{4} \{ {}_3 C_{t+\tau} (t + \tau) - {}_3 C_t t \}$$

$$= \frac{1}{6} \{ (t + \tau)^4 - t^4 \}.$$

Hyperprism of dimension n

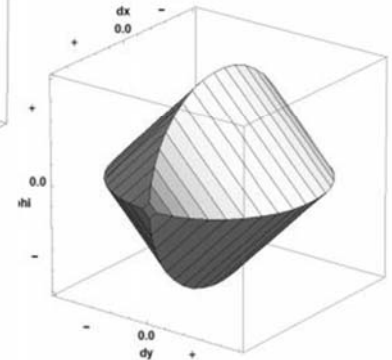
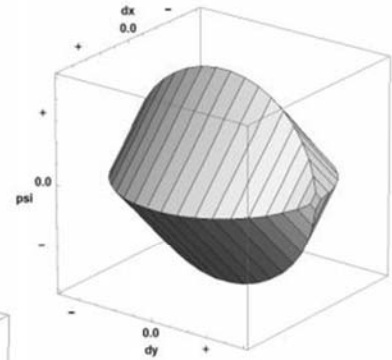
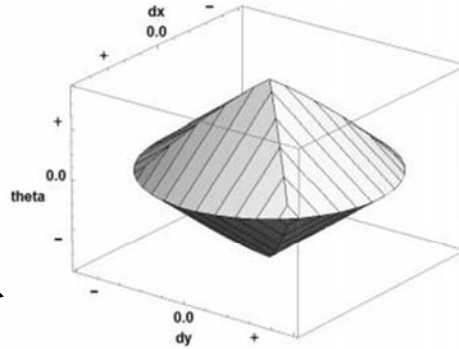
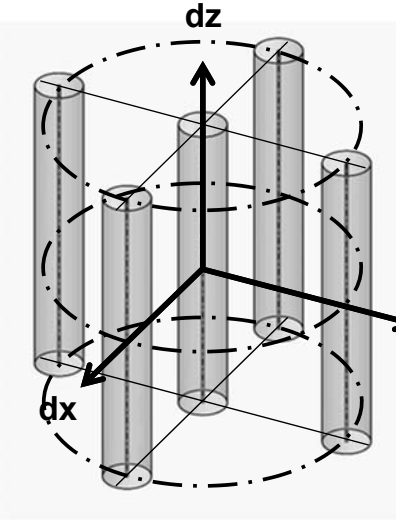
$${}_n V = {}_{n-1} C h$$

$${}_4 V_p = {}_3 C_t \tau = \frac{2}{3} t^3 \tau$$

Insight: if t = τ $V_{MMC} = 3.75 V_{RFS}$

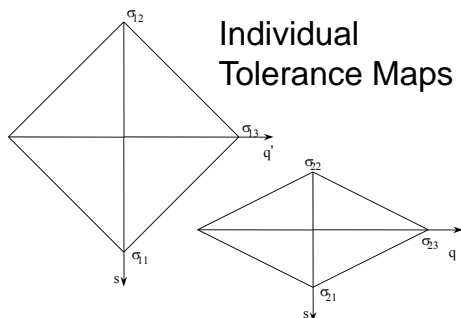
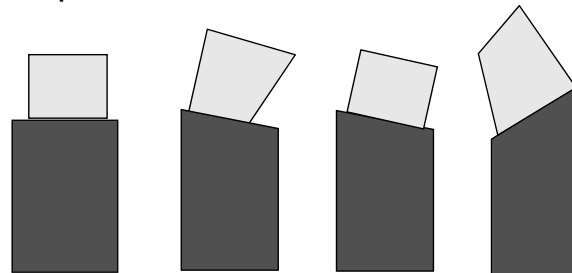
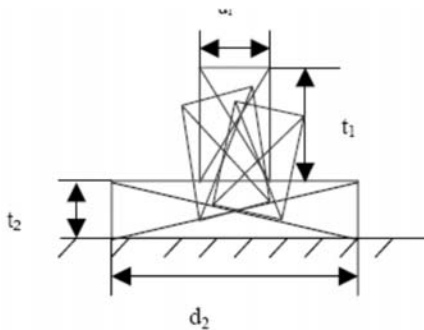
T-maps for Feature Patterns

- Pattern T-map is similar to Axis T-map but with one added dimension viz. theta.
- Theta represents possible rotation about dz axis.



Tolerance Analysis with T-maps: Minkowski Sums

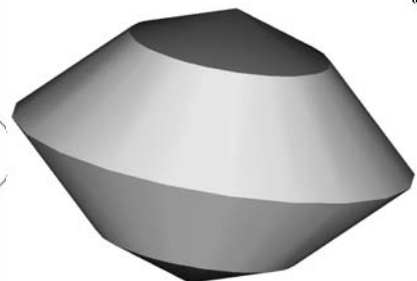
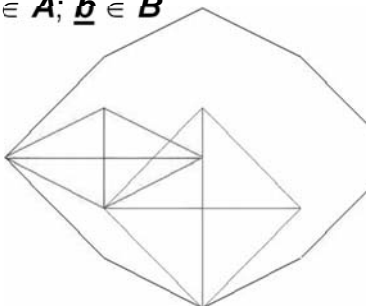
Variational possibilities – infinite combinations



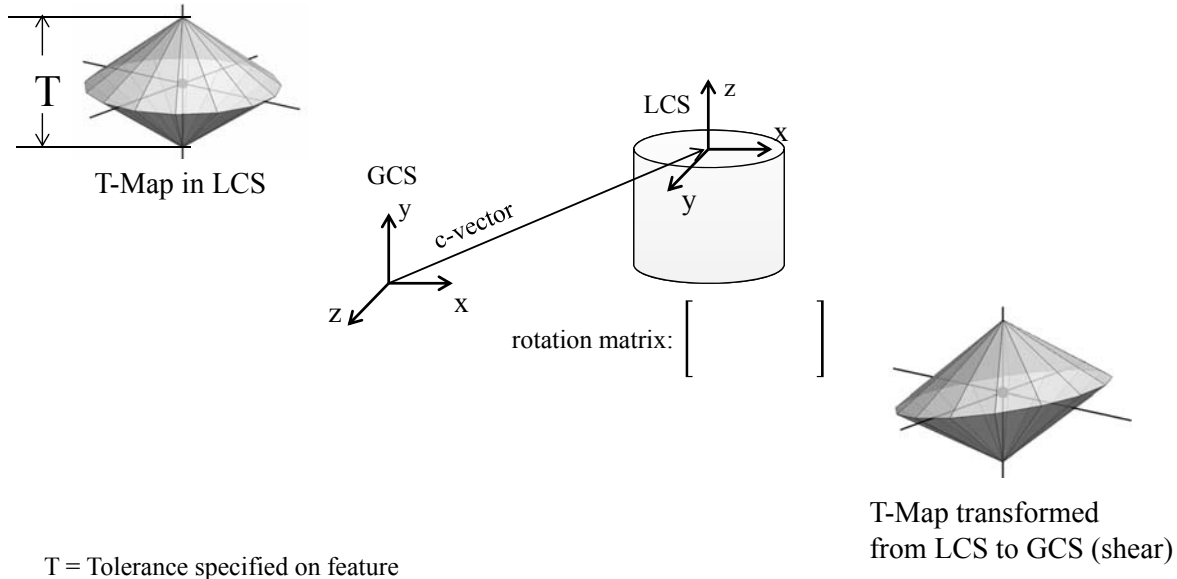
Individual Tolerance Maps

Minkowski sum: $C = U\mathbf{c}$, where $\mathbf{c} = \mathbf{a} + \mathbf{b}$ and $\mathbf{a} \in \mathbf{A}; \mathbf{b} \in \mathbf{B}$

Accumulation map

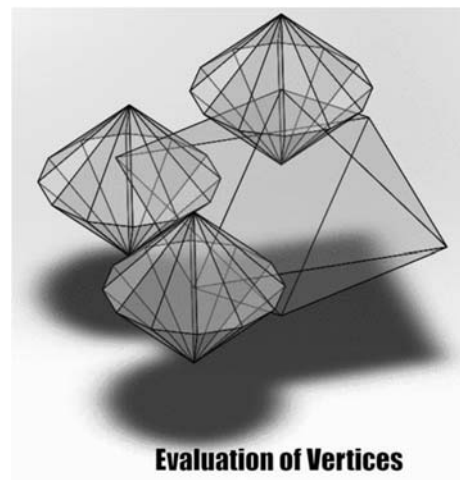


- Each feature's variation in the target gap is evaluated one by one by generating Local T-Map for that feature and then transforming it to Global coordinate frame



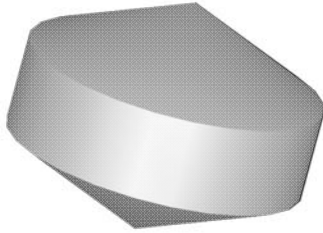
Analysis with T-maps: Minkowski sum

- **Minkowski Sum of two T-Maps A and B is a vector sum of its vertices in 6 dimensional space. One vertex is a six dimensional vector. Both T-Maps are represented with set of vectors.**
- Every vector in T-Map A is added to all vectors in T-Map B using vector sum operation.
- Minkowski Sum thus produced is represented as $A \oplus B$.
- Internal points so produced are not useful for further analysis and are omitted by forming a convex hull out of generated vectors.
- Further redundant vertices are eliminated using qhull to produce a convex hull. Which represents Minkowski Sum of two Tmaps.

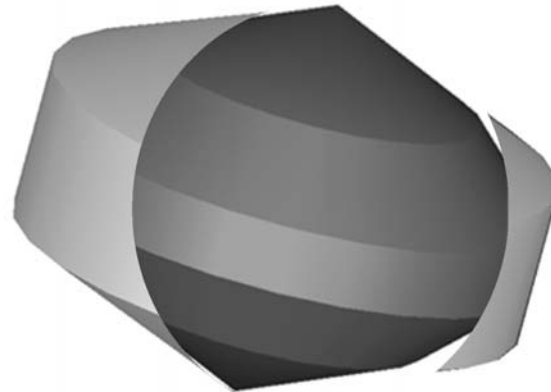


	T-map 1	T-map 2	Vector Addition	Reduced#
# Vertices	6	14	14 x 6 = 84	22

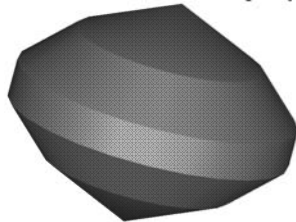
Functional Map
(e.g desired clearance)



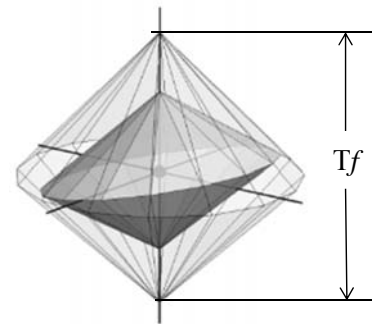
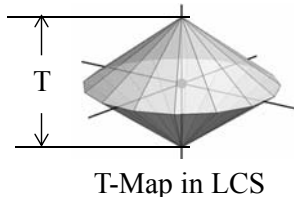
Does accumulation map fit inside functional map?



Accumulation Map (variations of all contributors : Minkowski sum of contributor T-maps)

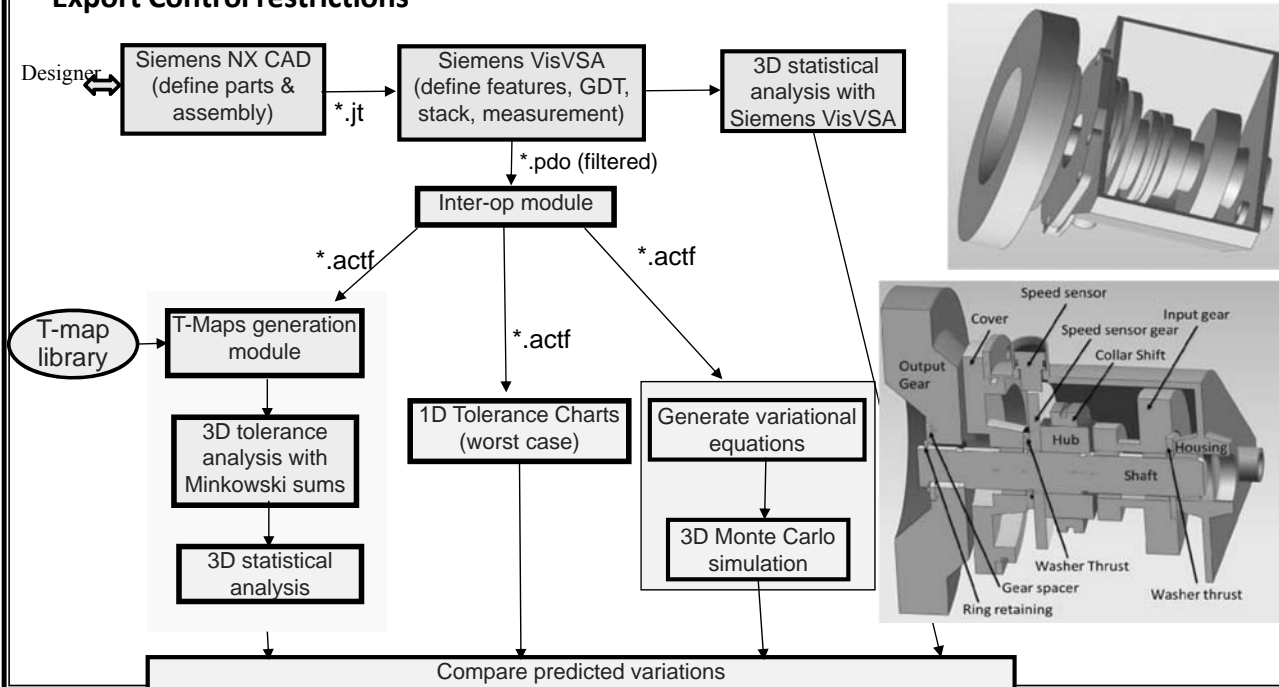


- Sensitivity of each feature to variation in target gap is evaluated one by one by generating Local T-Map for feature, transforming it to Global coordinate frame where target gap is located and then by fitting functional T-Map over Global T-Map.
- Sensitivity is ratio of tolerance associated with functional T-Map to the tolerance on feature

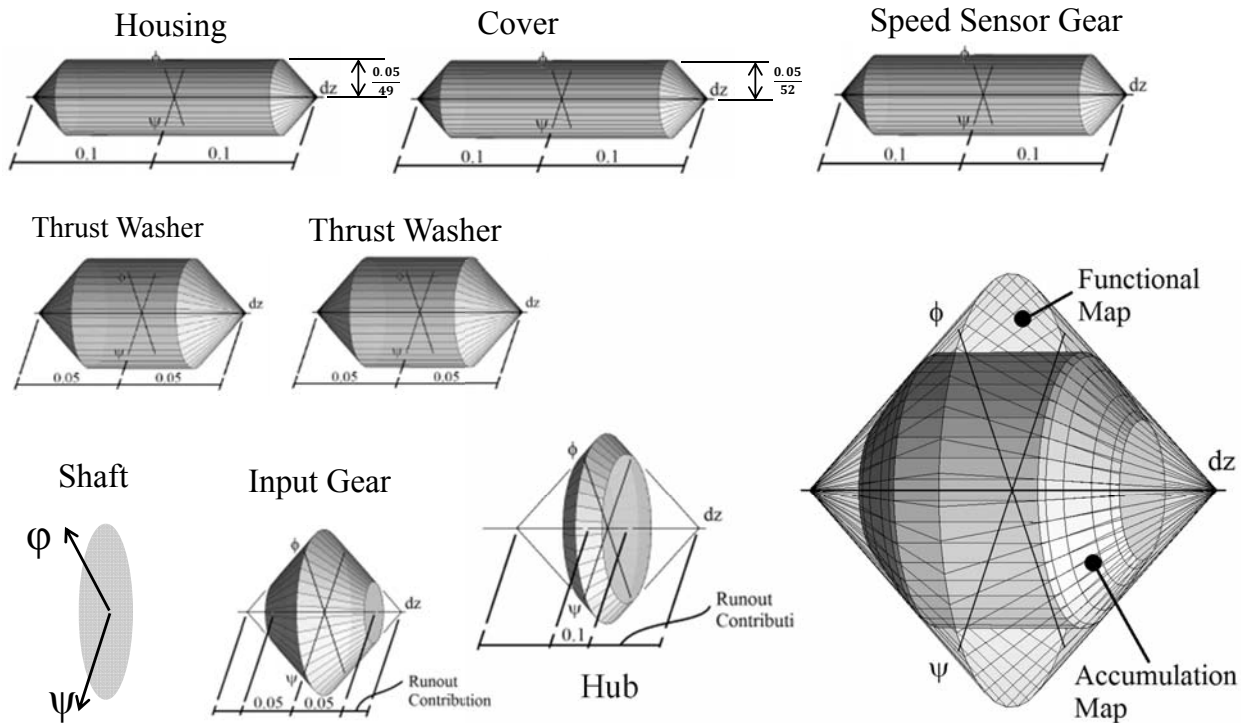


T = Tolerance specified on feature
 T_f = Functional tolerance at target gap contributed by feature
 S = Sensitivity of feature to target gap
 $= T_f / T$

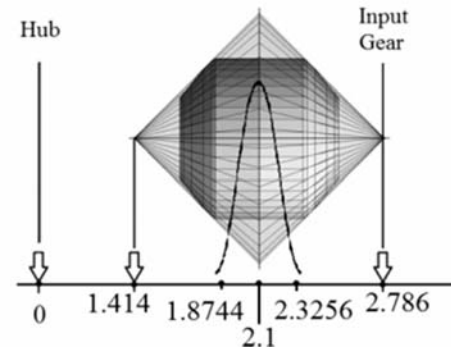
- T-map generation, transformation, accumulation procedures were applied to complex assemblies under a DMDII project
- This surrogate assembly will be used to illustrate the process because of RollsRoyce & Export Control restrictions



Worst Case analysis for Assembly T-Maps for features in stack for Gap 1



Sr.No.	Part	Tol. Type	Tol. Value T	Sensitivity S	Mean μ	SD (Six Sigma) σ
1	Hub	Size	0.10	1.03	0	0.0172
2	Shaft	Runout	0.10	1.52	0	0.0253
3	Speed Gear	Size	0.20	1.00	0	0.0333
4	Sensor	Parallelism	0.05	0.00	0	0.0000
5	Washer Thrust	Size	0.10	1.00	0	0.0167
6		Parallelism	0.05	0.00	0	0.0000
7	Cover	Size	0.20	1.00	0	0.0333
8		Parallelism	0.05	0.00	0	0.0000
9		Runout	0.10	0.13	0	0.0022
10	Housing	Size	0.20	1.00	0	0.0333
11		Parallelism	0.05	0.00	0	0.0000
12		Runout	0.09	0.13	0	0.0020
13	Input Gear	Size	0.10	1.19	0	0.0198
14		Runout	0.03	0.93	0	0.0046
15	Washer Thrust	Size	0.10	1.00	0	0.0167
16		parallelism	0.05	0.00	0	0.0000



```

### distribution parameters ###
-----
mean          -2.07456
variance      0.00193292
absolute deviation  0.0348872
skewness      0.0749801
kurtosis      0.182128
minimum value -2.24407
maximum value -1.8882
-----

```

$$V = \sum S * T$$

Where V = variation in clearance (Random variable)
 S = Sensitivity (Constant)
 T = Tolerance (Random variable)

$$V = 1.03T_1 + 1.52 T_2 + T_3 + T_7 + 0.13 T_9 + T_{10} + 0.13 T_{12} + 1.19 T_{13} + 0.93 T_{14} + T_{15}$$

- Models distinctly all types of tolerances including freeform profile
- Models tolerance variations in 3D
- Represents floating zones for orientation and form
- Accounts for material conditions: Models the tradeoff between size/orientation and position (bonus)
- Accounts for the choice and order of datum reference frames (DRFs)
- Can be used for worst case or statistical analysis
- Metrics can be used to aid in the allocation of tolerances in a stackup
- Is extendible to other features and to clusters of features
- All tolerance-Maps are convex bodies: allows use of some standard computational algorithms
- Is in Euclidean space: metric computations can be made using standard formulae